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ANALYSIS OF PRESSURE VARIATION OF FLUID IN BOUNDED CIRCULAR RESERVOIRS UNDER THE CONSTANT PRESSURE OUTER BOUNDARY CONDITION

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ABSTRACT

In this work, we have investigated the well pressure distribution in a bounded circular reservoir under the condition of constant pressure outer boundaries. The diffusivity equation was used in the analysis. The finite element technique, using Lagrange quadratic shape elements was employed to carry out the analysis over the cross-section of the reservoir which involves discretizing the domain into finite element, analysing these finite element, assembling the results from the analysis of the analysed finite element, imposing the boundary conditions and finally, getting the results that represent the entire domain. The results obtained where shown in a log log plot (dimensionless pressure against dimensionless time) for dimensionless radii of 1 to 1,000,000 in log cycles. It was shown that the relationship between dimensionless pressure and dimensionless time was linear whose slope was zero. The result obtained at the wellbore was compared with the results obtained by Van Everdigen and Hurst. It was shown that there was a strong positive correlation between the results. The result obtained from the analysis also shows the pressure variation outside wellbore of the same reservoir. It is important to note that solutions from existing literature only state the pressure at the wellbore at a particular time but this work predicts the pressure variation in the entire reservoir from the wellbore to the external boundary at the same time.

Keywords: Reservoir, Constant Terminal Rate, Dimensionless Variables, Diffusivity Equation, Wellbore And Weak Formulation.

Nomenclature w Weight function Initial reservoir pressure, psi P_{i} English Letters \forall For all Terminal flow rate Q *B* Formation volume factor, RB/STB *q* Volumetric flow rate, STB/D С Compressibility, psia⁻¹ Greek letters r Radius, ft *h* Thickness. ft Δt Time increment, hr r_D Dimensionless radius α Family of approximation *K* Stiffness matrix *M* Mass matrix Ø Porosity, fraction r External radius, ft *n* Number of elements k Permeability, md r_{eD} Dimensionless external radius *P* Pressure, psi μ Viscosity, cp r_{w} Wellbore radius, ft P_D Dimensionless pressure Pi π S Time step, hr ψ Interpolation function P_{D} Dimensionless pressure rate Time, hr t

 t_{D} Dimensionless time

1. INTRODUCTION

There are several methods of evaluating the reservoir parameters [1]. It was shown that solutions to differential equations describing flow in petroleum reservoir for given initial and boundary conditions can be expressed compactly using dimensionless variables and parameters. Several of these solutions are important in reservoir engineering applications [2–7]. Transient pressure response for a well producing from a finite reservoir of circular, square, and rectangular drainage shapes has been studied by [2, 8–14] among others. Everdingen And Hurst [2] presented the solution to the diffusivity equation in eq. (8) in the form of infinite series of exponential terms and Bessel functions. The authors evaluated this series for several values of r_{eD} even a wide represented the solution for the function of the series of the series of the series of the series for several values of r_{eD} even a wide represented the series [15] and Well

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over a wide range of values for t_D . Chatas [15] and Well

[16] conveniently tabulated these solutions for the following two cases: Infinite-acting reservoir and Finite-radial reservoir.

Mishra and Ramey [17] presented a build-up derivative type curve for a well with storage and skin, and producing from the centre of a closed, circular reservoir. Their type-curve applies for large producing times. The work by [18] presents drawdown and build-up pressure derivative type-curves for a well producing at a constant rate from the centre of a finite, circular reservoir. The outer boundary may be closed, or at a constant pressure. The differences between the responses for a well in a closed, circular reservoir (fully developed field), and a well in a circular reservoir with a constant-pressure outer boundary (active edge water drive system, or developed five-spot fluid-injection pattern) were discussed. Design relations were developed to estimate the time period which corresponds to infinite-acting radial flow, or to a semi-log straight line on a pressure vs. logarithm of time graph. Producing time effects on buildup responses were studied using the slope of a dimensionless build-up graph proposed in [19].

In all the literatures reviewed so far, the researchers focused on predicting the wellbore pressure [2, 15, 16], etc. Sometimes, it is important to know the pressure history outside the reservoir is scarce in the literature. This study therefore seeks to look at the reservoir pressure both within and outside the wellbore of a bounded circular reservoir.

2. THEORY

The law of conservation of mass, Darcy's law and the equation of state has been combined to obtain the following partial differential equation:

$$\frac{\partial^2 P}{\partial r^2} + \frac{1}{r} \frac{\partial P}{\partial r} = \frac{\phi \mu c}{0.000264k} \frac{\partial P}{\partial t} \qquad (1)$$

with the assumptions that compressibility, c is small and independent of pressure, P; permeability, k, is constant and isotropic; viscosity, , μ is independent of pressure; porosity, , ϕ is constant; and that certain terms in the basic differential equation (involving pressure gradients squared) are negligible. This equation is called the diffusivity equation and the term $\frac{\phi\mu c}{0.000264k}$ is the inverse of the diffusivity constant,. μ

In this work, the diffusivity equation was analysed for bounded circular reservoirs, the case in which the well is assumed to be located in the centre of a cylindrical reservoir under the condition of constant external boundary.

3. GOVERNING EQUATION

$$\frac{\partial^2 P}{\partial r^2} + \frac{1}{r} \frac{\partial P}{\partial r} = \frac{\phi \mu c}{0.000264k} \frac{\partial P}{\partial t}$$
(1)

Initial and boundary conditions:

i.
$$P = P_i \operatorname{att} = 0 \forall r$$
 (2)

ii.
$$\left(r\frac{\partial P}{\partial r}\right)_{r_W} = \frac{q^D \mu}{2\pi kh} \text{ for } > .0$$
 (3)

iii.
$$\left(r\frac{\partial P}{\partial r}\right)_{r_W} = 0 \ \forall \ t$$
 (4)

The above equations incorporate physical parameters such as permeability, and it would be futile to solve this problem for a particular combination of values for these parameters. Dimensionless variables are designed to eliminate the physical parameters that affect quantitatively, but not qualitatively, the reservoir response. The above equations are in Darcy units, and the dimensionless terms will render the system of units employed irrelevant. For this line source model, 3 dimensionless variables are required. In US Oilfield units, distance, time and pressure are replaced as follows: Dimensionless time:

$$t_D = \frac{0.0002637kt}{\phi \mu c r_w^2}$$
(5)

Dimensionless distance:

$$r_D = \frac{r}{r_w} \tag{6}$$

Dimensionless pressure:

$$P_D = \frac{kh}{141.2_q B_\mu} (P_1 - P) \tag{7}$$

By defining dimensionless variables this way, it is possible to create an analytical model of the well and reservoir, or theoretical 'type-curve', that provides a 'global' description of the pressure response that is independent of the flow rate or actual values of the well and reservoir parameters.

Eq.1 can be transformed by substituting the following dimensionless variables in Eqs. 5-7 into eq. 1 and this becomes:

$$\frac{\partial^2 P_D}{\partial r_D^2} + \frac{1}{r_D} \frac{\partial P_D}{\partial r_D} = \frac{\partial P_D}{\partial t_D}$$
(8)

and the initial and boundary conditions becomes:

1. Dimensionless initial condition (uniform pressure in the reservoir):

$$P_D(r_D, t_D = 0) \tag{9}$$

 Dimensionless inner boundary condition (constant rate at the wellbore):

$$\frac{\partial P_D}{\partial r_D}(1, t_D) = -1 \tag{10}$$

3. Dimensionless Outer Boundary Conditions: Constant pressure outer boundary

$$P_D = (r_{eD}, t_D) = 0$$
 (11)

Eq. 8 can also be written in a condensed form as:

$$\frac{1}{r_D}\frac{\partial}{\partial r_D}\left(r_D\frac{\partial P_D}{\partial r_D}\right) = \frac{\partial P_D}{\partial t_D}$$
(12)

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4. FINITE ELEMENT FORMULATION

4.1 Weak Formulation

In the development of the weak form, we assumed a quadratic element mesh and placed it over the domain and apply the following steps:

From eq. 12, we have:

$$\frac{\partial P_D}{\partial t_D} - \frac{1}{r_D} \frac{\partial}{\partial r_D} \left(r_D \frac{\partial P_D}{\partial r_D} \right) = 0$$
(13)

Multiply eq. 13 by the weight W function and integrate the final equation over the domain.

$$\int_{v} w \left[\frac{\partial P_D}{\partial t_D} - \frac{1}{r_D} \frac{\partial}{\partial r_D} \left(r_D \frac{\partial P_D}{\partial r_D} \right) \right] dv = 0 \quad (14)$$

Eq. 14 becomes,

$$\int_{0}^{1} \int_{0}^{2\pi r} \int_{r_{DA}}^{DB} w \left[\frac{\partial P_D}{\partial t_D} - \frac{1}{r_D} \frac{\partial}{\partial r_D} \left(r_D \frac{\partial P_D}{\partial r_D} \right) \right] r_D dr_D d\theta dz = 0$$
(15)

Integrating eq. 15 with respect to z, then, θ over the limits, we have:

$$\int_{r_{DA}}^{r_{DB}} w \left[\frac{\partial P_D}{\partial t_D} + \frac{1}{r_D} \frac{\partial}{\partial r_D} \left(r_D \frac{\partial P_D}{\partial r_D} \right) \right] r_D dr_D = 0 \quad (16)$$

Eq. 16 can be exploded into:

$$\int_{DA}^{r_{DB}} w \frac{\partial P_D}{\partial t_D} r_D dr_D - \int_{r_{DA}}^{r_{DB}} w \frac{\partial}{\partial r_D} \left(r_D \frac{\partial P_D}{\partial r_D} \right) dr_D = 0 \quad (17)$$

Integrating eq. 17 by part, we have:

$$\int_{r_{DA}}^{r_{DB}} r_D \frac{\partial w}{\partial r_D} \frac{\partial P_D}{\partial r_D} dr_D - w \left[r_D \frac{\partial P_D}{\partial r_D} \right]_{r_{DA}}^{r_{DB}} + \int_{r_{DA}}^{r_{DB}} r_D w \frac{\partial P_D}{\partial t_D} dr_D$$
$$= 0$$
(18)

Grouping eq. 18 into linear and bilinear components, we have:

$$\int_{r_{DA}}^{r_{DB}} r_D \frac{\partial w}{\partial r_D} \frac{\partial P_D}{\partial r_D} dr_D + \int_{r_{DA}}^{r_{DB}} r_D w \frac{\partial P_D}{\partial t_D} dr_D - w \left[r_D \frac{\partial P_D}{\partial r_D} \right]_{r_{DA}}^{r_{DB}}$$

$$= 0 \tag{19}$$

$$\int_{r_{DA}} r_D \frac{\partial w}{\partial r_D} \frac{\partial P_D}{\partial r_D} dr_D + \int_{r_{DA}} r_D w \frac{\partial P_D}{\partial t_D} dr_D - wQ_A - wQ_B$$
$$= 0 \quad (20)$$
Where $Q = r_D \frac{\partial P_D}{\partial t_D}$

Where $Q = r_D \frac{\partial P_D}{\partial r_D}$

5. INTERPOLATION FUNCTION

The weak form in eq. 20 requires that the approximation chosen for P_D should be at least quadratic in r_D so that there are no terms in eq. 20 that are identically zero. Since the primary variable is simply the function itself, the Lagrange family of interpolation functions is admissible. We proposed that P_D is the approximation over a typical finite element domain by the expression:

$$P_D(r_D, t_D) = \sum_{j=1}^n P_{Dj}(t_D) \psi_j^e(r_D) \text{ and } w = \psi_j^e(r_D) \quad (21)$$

Substituting eq. 21 into eq. 20, we have:

$$\int_{r_{DA}}^{r_{DB}} r_{D} \frac{d\psi_{j}^{e}}{dr_{D}} \frac{\partial}{dr_{D}} \sum_{j=1}^{n} P_{Dj}(t_{D})\psi_{j}^{e}(r_{D})dr_{D} + \int_{r_{DA}}^{r_{DB}} r_{D}\psi_{j}^{e} \frac{d}{dt_{D}} \sum_{j=1}^{n} P_{Dj}(t_{D})\psi_{j}^{e}(r_{D})dr_{D} - Q_{i}^{e} = 0$$
(22)

Factor out $\sum_{j=1}^{n} P_{Dj}$

$$\sum_{j=1}^{n} P_{Dj} + \int_{r_{DA}}^{r_{DB}} r_{D} \frac{d\psi_{i}^{e}}{dr_{D}} \frac{d\psi_{j}^{e}}{dr_{D}} dr_{D} + \sum_{j=1}^{n} \dot{P}_{Dj} \int_{r_{DA}}^{r_{DB}} r_{D} \psi_{i}^{e} \psi_{j}^{e} dr_{D} - Q_{i}^{e} = 0$$
(23)

where
$$\dot{P}_{Dj} = \frac{dT_D}{dt_D}$$

In matrix form we can represent the semi-discrete finite element model as thus,

$$|K_{ij}^{e}|\{P_{D}\} + |M_{ij}^{e}|\{\dot{P}_{Dj}\} = \{Q_{i}^{e}\} \quad (24)$$

Where

$$K_{ij}^{e} = \int_{\substack{r_{DA} \\ r_{DB}}}^{r_{DB}} r_{D} \frac{d\psi_{i}^{e}}{dr_{D}} \frac{d\psi_{j}^{e}}{dr_{D}} dr_{D}$$
(25)

$$M_{ij}^e = \int_{r_{DA}}^{DB} r_D \,\psi_i^e \psi_j^e dr_D \tag{26}$$

Using Quadratic Lagrange Interpolation functions for a quadratic element:

$$\psi_1(r) = \frac{1}{h^2}(h + r_A - r)(h - 2r + 2r_A)$$
(27)

$$\psi_2(r) = \frac{4}{h^2}(r - r_A)(h + r_A - r)$$
(28)

$$\psi_3(r) = \frac{-1}{h^2} (r - r_A) \left(h - 2r + 2r_A \right)$$
(29)

The coefficient matrix can be easily derived by substituting the Lagrange interpolation functions into eq. 25 respectively. The matrices are shown below: $[K^e]$

$$=\frac{1}{6h} \begin{bmatrix} 3h+14r_A & -(4h+16r_A) & h+2r_A \\ -(4h+16r_A) & 16h+32r_A & -(12h+16r_A) \\ h+2r_A & -(12h+16r_A) & 11h+14r_A \end{bmatrix} (30)$$

Also, the mass matrices can be easily derived by substituting the Lagrange interpolation functions into eq. 26 respectively. The matrices are shown below:

$$[M^{e}] = \frac{1}{60} \begin{bmatrix} h + 8r_{A} & 4r_{A} & -h - 2r_{A} \\ 4r_{A} & 16h + 32r_{A} & 4h + 4r_{A} \\ -h - 2r_{A} & 4h + 4r_{A} & 7h + 8r_{A} \end{bmatrix}$$
(31)

Using four quadratic elements,

$$r_A = r_w + (n-1)h$$
 (32)

In this analysis, we have withheld the computational details of the shape assembly of the finite element

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analysis (FEA) used. However, the authors would be glad to interact with researchers who may want to refer to the computational mathematics involved.

6. TIME APPROXIMATION

For a given time step s, eq. 24 will be written as

$$\left|K_{ij}^{e}\right|\{P_{D}\} + \left|M_{ij}^{e}\right|\{\dot{P}_{Dj}\} = \{Q_{i}^{e}\}$$
(33)

For the next time step s+1, eq. 24 becomes

$$\left|K_{ij}^{e}|\{P_{D}\}_{s+1} + \left|M_{ij}^{e}\right|\{\dot{P}_{Dj}\}_{s+1} = \{Q_{i}^{e}\}_{s+1}$$
(34)

Multiply eq.33 by $(1-\alpha)$ and eq. 34 by α , then we add the two resulting equations,

$$\begin{bmatrix} M_{ij}^{e} \end{bmatrix} \begin{bmatrix} (1-a) \{ \dot{P}_{Dj} \}_{s}^{} + a \{ \dot{P}_{Dj} \}_{s+1} \end{bmatrix} \\ + \begin{bmatrix} K_{ij}^{e} \end{bmatrix} \begin{bmatrix} (1-a) \{ P_{Dj} \}_{s}^{} + a \{ P_{Dj} \}_{s+1} \end{bmatrix} \\ = (1-a) \{ Q_{i}^{e} \}_{s}^{} + \{ Q_{i}^{e} \}_{s+1}$$
(35)

The α family of interpolation for time consideration is given as:

$$(1-a)\{\dot{P}_{Dj}\}_{s} + a\{\dot{P}_{Dj}\}_{s+1} = \frac{\{P_{Dj}\}_{s+1} - \{P_{Dj}\}_{s}}{\Delta t_{s+1}} \quad (36)$$

Substitute eq.36 into eq.35 and using the Crank-Nicholson Scheme where $\alpha = 1/2$,

$$\begin{bmatrix} \left[M_{ij}^{e} \right] + \frac{\Delta t_{s+1}}{2} \left[K_{ij}^{e} \right] \right] \left\{ P_{Dj} \right\}_{s+1} \\ = \begin{bmatrix} \left[M_{ij}^{e} \right] - \frac{\Delta t_{s+1}}{2} \left[K_{ij}^{e} \right] \right] \left\{ P_{Dj} \right\}_{s} \\ + \frac{\Delta t_{s+1}}{2} \left[\left\{ Q_{i}^{e} \right\}_{s} + \left\{ Q_{i}^{e} \right\}_{s+1} \right]$$
(37)

From the initial condition given in eq. 9 for a constant terminal rate case, it implies that when s = 0, all dimensionless pressure in the reservoir will be zero. Also, the flow rate was constant at the wellbore all through operation. This means that $[\{Q_i^e\}_{s+1}]$.Hence, eq. 37 becomes:

$$\left| \begin{bmatrix} M_{ij}^e \end{bmatrix} + \frac{\Delta t_{s+1}}{2} \begin{bmatrix} K_{ij}^e \end{bmatrix} \right| \left\{ P_{Dj} \right\}_1$$
$$= \left[\begin{bmatrix} M_{ij}^e \end{bmatrix} - \frac{\Delta t_1}{2} \begin{bmatrix} K_{ij}^e \end{bmatrix} \left\{ P_{Dj} \right\}_0$$
$$+ \Delta t_1 \{ \bar{Q} \stackrel{e}{i} \} \right]$$
(38)

Where
$$\bar{Q}_{i}^{e} = \frac{1}{2} (Q_{i}^{1})_{s+1} + \frac{1}{2} (Q_{i}^{1})_{s}$$

 $\{P_{Dj}\}_{1} = \left[\left[M_{ij}^{e} \right] + \frac{\Delta t_{1}}{2} \left[K_{ij}^{e} \right] \right]^{-1} \left[\left[M_{ij}^{e} \right] - \frac{\Delta t_{1}}{2} \left[K_{ij}^{e} \right] \right] \{P_{Dj}\}_{0}$
 $+ \Delta t_{1} \{ \bar{Q}_{i}^{e} \}$ (39)

7. RESULTS AND DISCUSSION

The steady state condition applies, after the transient period, to a well-draining a cell which has a completely open outer boundary. It is assumed that, for a constant rate of production, fluid withdrawal from the well will be exactly balanced by fluid entry across the outer boundary and therefore,

$$P = P_e = \text{Constant at } r \text{ i. e. , } P_D(r_D \to r_{eD}, t_D) = 0 \text{ and } \frac{\partial P}{\partial t}$$
$$= 0 \forall r \text{ and } t \forall$$

This condition is appropriate when pressure is being maintained in the reservoir due to either natural water influx or artificially by the injection of some displacing fluid. The semi-steady state flow equations are frequently applied when the rate, and consequently the position of the no-flow boundary surrounding a well, is slowly varying functions of time. If the production rate of an individual well is changed, for instance, due to closure for repair or increasing the rate to obtain a more even fluid withdrawal pattern throughout the reservoir, there will be a brief period when transient flow conditions prevail followed by stabilized flow for the new distribution of individual well rates.

Thus, this solution of the diffusivity equation models radial flow of slightly compressible liquid in a homogeneous reservoir of uniform thickness; reservoir at uniform pressure before production; unchanging pressure at the outer boundary; and production at constant rate from a single well (centred in the reservoir) with wellbore radius.

The results obtained from this analysis were shown in the form of graphs of dimensionless pressure against dimensionless time. This was shown in Fig. 1. Fig. 1 is a log log plot of dimensionless pressure against dimensionless time. The graph shows for different dimensionless radii ranging between 1 and 1000000 in log cycles. It was seen from the graph that the dimensionless pressure history of the reservoir was not captured at the initial stage between the dimensionless time of zero and the respective dimensionless times in Fig. 1. This was due to the fact that, within these regions, the reservoir was at the infinite acting state. After these infinite acting period, it was observed that the dimensionless pressure increases and later becomes uniform became the withdrawn fluid has been completely replaced. This condition remains throughout the entire life of the reservoir presumably.

To test for the degree of accuracy of the results, a percentage error computation was done between the FEM and the results published by [2]. Table 1 shows the percentage error between the FEM solutions and the Van Everdigen and Hurst solutions to show the level of discrepancies between the two results. It was shown that there was a strong correlation between the two results.



Fig. 1: Log-Log plot of PD against tD for various r_D *in the infinite acting regime*

| $r_{eD} = 1$ | .5, n = 4 | $r_{eD} =$ | 2, <i>n</i> = 4 | $r_{eD} = 2$ | 2.5, n = 4 | $r_{eD} =$ | 3, <i>n</i> = 4 | $r_{eD} =$ | 3.5, n = 4 | $r_{eD} =$ | 4, <i>n</i> = 4 | $r_{eD} =$ | = 6, <i>n</i> = 4 |
|--------------|-----------|------------|-----------------|--------------|------------|------------|-----------------|------------|------------|------------|-----------------|------------|-------------------|
| t_D | % error | t_D | % error | t_D | % error | t_D | % error | t_D | % error | t_D | % error | t_D | % error |
| 0.05 | 0.4348 | 0.2 | 0.0000 | 0.3 | 0.0000 | 0.5 | 0.1621 | 0.5 | 0.4839 | 1 | 0.0000 | 4 | 0.2353 |
| 0.055 | 0.4167 | 0.22 | 0.0000 | 0.35 | 0.0000 | 0.55 | 0.1563 | 0.6 | 0.4511 | 1.2 | 0.1167 | 4.5 | 0.1515 |
| 0.06 | 0.0000 | 0.24 | 0.0000 | 0.4 | 0.0000 | 0.6 | 0.0000 | 0.7 | 0.4255 | 1.4 | 0.1105 | 5 | 0.1470 |
| 0.07 | 0.0000 | 0.26 | 0.0000 | 0.45 | 0.0000 | 0.7 | 0.0000 | 0.8 | 0.2699 | 1.6 | 0.0000 | 5.5 | 0.1431 |
| 0.08 | 0.0000 | 0.28 | 0.0000 | 0.5 | 0.0000 | 0.8 | 0.0000 | 0.9 | 0.3876 | 1.8 | 0.1014 | 6 | 0.1397 |
| 0.09 | 1.0274 | 0.3 | 0.0000 | 0.55 | 0.0000 | 0.9 | 0.0000 | 1 | 0.2488 | 2 | 0.0000 | 6.5 | 0.1368 |
| 0.1 | 0.0000 | 0.35 | 0.0000 | 0.6 | 0.0000 | 1 | 0.0000 | 1.2 | 0.2331 | 2.2 | 0.0951 | 7 | 0.1342 |
| 0.12 | 0.0000 | 0.4 | 0.0000 | 0.7 | 0.0000 | 1.2 | 0.0000 | 1.4 | 0.1106 | 2.4 | 0.0000 | 7.5 | 0.1319 |
| 0.14 | 0.0000 | 0.45 | 0.1748 | 0.8 | 0.1374 | 1.4 | 0.0000 | 1.6 | 0.1058 | 2.6 | 0.5425 | 8 | 0.1300 |
| 0.16 | 0.0000 | 0.5 | 0.0000 | 0.9 | 0.1325 | 1.6 | 0.1079 | 1.8 | 0.1019 | 2.8 | 0.0000 | 8.5 | 0.1281 |
| 0.18 | 0.0000 | 0.55 | 0.0000 | 1 | 0.0000 | 1.8 | 0.0000 | 2 | 0.0988 | 3 | 0.0000 | 9 | 0.1266 |
| 0.2 | 0.0000 | 0.6 | 0.0000 | 1.2 | 0.1227 | 2 | 0.0000 | 2.2 | 0.1921 | 3.4 | 0.0000 | 10 | 0.1238 |
| 0.22 | 0.0000 | 0.65 | 0.0000 | 1.4 | 0.0000 | 2.2 | 0.0000 | 2.4 | 0.0939 | 3.8 | 0.0000 | 12 | 0.1200 |
| 0.24 | 0.0000 | 0.7 | 0.0000 | 1.6 | 0.0000 | 2.4 | 0.0000 | 2.6 | 0.0920 | 4.5 | 0.0000 | 14 | 0.1174 |
| 0.26 | 0.0000 | 0.75 | 0.0000 | 1.8 | 0.0000 | 2.6 | 0.0000 | 2.8 | 0.0904 | 5 | 0.0000 | 16 | 0.0578 |
| 0.28 | 0.0000 | 0.8 | 0.0000 | 2 | 0.0000 | 2.8 | 0.0000 | 3 | 0.0000 | 5.5 | 0.0000 | 18 | 0.1144 |
| 0.3 | 0.0000 | 0.85 | 0.0000 | 2.2 | 0.0000 | 3 | 0.0000 | 3.5 | 0.0864 | 6 | 0.0755 | 20 | 0.1135 |
| 0.35 | 0.0000 | 0.9 | 0.0000 | 2.4 | 0.0000 | 3.5 | 0.0000 | 4 | 0.0845 | 7 | 0.0742 | 22 | 0.1129 |
| 0.4 | 0.0000 | 0.95 | 0.0000 | 2.6 | 0.0000 | 4 | 0.0000 | 5 | 0.0000 | 8 | 0.0000 | 24 | 0.1125 |
| 0.45 | 0.0000 | 1 | 0.0000 | 2.8 | 0.0000 | 4.5 | 0.0000 | 6 | 0.0000 | 9 | 0.0000 | 26 | 0.1123 |
| 0.5 | 0.0000 | 1.2 | 0.0000 | 3 | 0.0000 | 5 | 0.0000 | 7 | 0.0000 | 10 | 0.0000 | 28 | 0.0561 |

Table 1: Percentage error of FEM and Van Everdigen

| rD | | | | | | | | | |
|-------|-------|--------|--------|--------|--------|--------|--------|--------|--------|
| tD | 1 | 1.0625 | 1.1250 | 1.1875 | 1.2500 | 1.3125 | 1.3750 | 1.4375 | 1.5000 |
| 0.050 | 0.229 | 0.174 | 0.127 | 0.091 | 0.063 | 0.041 | 0.025 | 0.011 | 0.000 |
| 0.055 | 0.241 | 0.183 | 0.137 | 0.099 | 0.070 | 0.046 | 0.028 | 0.013 | 0.000 |
| 0.060 | 0.249 | 0.192 | 0.145 | 0.107 | 0.076 | 0.051 | 0.031 | 0.015 | 0.000 |
| 0.070 | 0.266 | 0.209 | 0.160 | 0.121 | 0.087 | 0.060 | 0.037 | 0.018 | 0.000 |
| 0.080 | 0.282 | 0.224 | 0.174 | 0.133 | 0.098 | 0.068 | 0.043 | 0.020 | 0.000 |
| 0.090 | 0.295 | 0.237 | 0.187 | 0.144 | 0.107 | 0.075 | 0.048 | 0.023 | 0.000 |
| 0.100 | 0.307 | 0.249 | 0.198 | 0.154 | 0.115 | 0.082 | 0.052 | 0.025 | 0.000 |
| 0.120 | 0.328 | 0.269 | 0.216 | 0.170 | 0.129 | 0.092 | 0.059 | 0.029 | 0.000 |
| 0.140 | 0.344 | 0.284 | 0.231 | 0.183 | 0.140 | 0.101 | 0.065 | 0.032 | 0.000 |
| 0.160 | 0.356 | 0.297 | 0.243 | 0.194 | 0.149 | 0.108 | 0.069 | 0.034 | 0.000 |
| 0.180 | 0.367 | 0.307 | 0.252 | 0.202 | 0.156 | 0.113 | 0.073 | 0.036 | 0.000 |
| 0.200 | 0.375 | 0.315 | 0.260 | 0.209 | 0.161 | 0.117 | 0.076 | 0.037 | 0.000 |
| 0.220 | 0.381 | 0.321 | 0.265 | 0.214 | 0.166 | 0.121 | 0.078 | 0.038 | 0.000 |
| 0.240 | 0.386 | 0.326 | 0.270 | 0.218 | 0.169 | 0.123 | 0.080 | 0.039 | 0.000 |
| 0.260 | 0.390 | 0.330 | 0.274 | 0.221 | 0.172 | 0.125 | 0.082 | 0.040 | 0.000 |
| 0.280 | 0.393 | 0.333 | 0.277 | 0.224 | 0.174 | 0.127 | 0.083 | 0.040 | 0.000 |
| 0.300 | 0.396 | 0.335 | 0.279 | 0.226 | 0.176 | 0.128 | 0.084 | 0.041 | 0.000 |
| 0.350 | 0.400 | 0.340 | 0.283 | 0.229 | 0.179 | 0.131 | 0.085 | 0.042 | 0.000 |
| 0.400 | 0.402 | 0.342 | 0.285 | 0.231 | 0.180 | 0.132 | 0.086 | 0.042 | 0.000 |
| 0.450 | 0.404 | 0.343 | 0.286 | 0.232 | 0.181 | 0.133 | 0.086 | 0.042 | 0.000 |
| 0.500 | 0.405 | 0.344 | 0.287 | 0.233 | 0.182 | 0.133 | 0.087 | 0.042 | 0.000 |
| 0.600 | 0.405 | 0.345 | 0.287 | 0.233 | 0.182 | 0.133 | 0.087 | 0.043 | 0.000 |
| 0.800 | 0.405 | 0.345 | 0.288 | 0.234 | 0.182 | 0.134 | 0.087 | 0.043 | 0.000 |

Table 2: Dimensionless Pressure Distribution for $r_{eD}=1.5$, n=4 and $\Delta t=0.005$

Table 3: Dimensionless Pressure Distribution for $r_{eD} = 10$, n = 4 and $\Delta t = 0.05$

| ∕_ rD | | | | | | | | | |
|-------|--------|--------|--------|--------|--------|--------|--------|--------|---------|
| tD | 1.0000 | 2.1250 | 3.2500 | 4.3750 | 5.5000 | 6.6250 | 7.7500 | 8.8750 | 10.0000 |
| 10 | 1.639 | 0.923 | 0.547 | 0.327 | 0.191 | 0.107 | 0.055 | 0.023 | 0.000 |
| 12 | 1.719 | 0.999 | 0.615 | 0.384 | 0.235 | 0.139 | 0.076 | 0.032 | 0.000 |
| 14 | 1.719 | 0.999 | 0.615 | 0.384 | 0.235 | 0.139 | 0.076 | 0.032 | 0.000 |
| 16 | 1.845 | 1.120 | 0.725 | 0.479 | 0.311 | 0.195 | 0.112 | 0.050 | 0.000 |
| 18 | 1.896 | 1.170 | 0.770 | 0.518 | 0.343 | 0.219 | 0.128 | 0.058 | 0.000 |
| 20 | 1.941 | 1.213 | 0.811 | 0.553 | 0.372 | 0.241 | 0.142 | 0.065 | 0.000 |
| 25 | 2.032 | 1.302 | 0.892 | 0.624 | 0.430 | 0.285 | 0.171 | 0.079 | 0.000 |
| 30 | 2.099 | 1.367 | 0.952 | 0.676 | 0.474 | 0.318 | 0.193 | 0.089 | 0.000 |
| 35 | 2.149 | 1.415 | 0.996 | 0.715 | 0.506 | 0.342 | 0.209 | 0.097 | 0.000 |
| 40 | 2.186 | 1.451 | 1.029 | 0.744 | 0.530 | 0.360 | 0.221 | 0.103 | 0.000 |
| 45 | 2.213 | 1.477 | 1.054 | 0.766 | 0.547 | 0.374 | 0.230 | 0.107 | 0.000 |
| 50 | 2.234 | 1.497 | 1.072 | 0.782 | 0.560 | 0.383 | 0.236 | 0.110 | 0.000 |
| 55 | 2.249 | 1.511 | 1.085 | 0.793 | 0.570 | 0.391 | 0.241 | 0.113 | 0.000 |
| 60 | 2.260 | 1.522 | 1.095 | 0.802 | 0.577 | 0.396 | 0.245 | 0.114 | 0.000 |
| 65 | 2.268 | 1.530 | 1.103 | 0.809 | 0.583 | 0.400 | 0.247 | 0.116 | 0.000 |
| 70 | 2.274 | 1.536 | 1.108 | 0.813 | 0.587 | 0.403 | 0.249 | 0.117 | 0.000 |

| ∕rD | | | | | | | | | |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|---------|
| tD | 1.0000 | 2.1250 | 3.2500 | 4.3750 | 5.5000 | 6.6250 | 7.7500 | 8.8750 | 10.0000 |
| 80 | 2.282 | 1.544 | 1.115 | 0.819 | 0.592 | 0.407 | 0.252 | 0.118 | 0.000 |
| 90 | 2.286 | 1.548 | 1.119 | 0.823 | 0.594 | 0.409 | 0.253 | 0.119 | 0.000 |
| 100 | 2.289 | 1.550 | 1.121 | 0.825 | 0.596 | 0.410 | 0.254 | 0.119 | 0.000 |
| 110 | 2.290 | 1.552 | 1.122 | 0.826 | 0.597 | 0.411 | 0.254 | 0.119 | 0.000 |
| 120 | 2.291 | 1.552 | 1.123 | 0.826 | 0.597 | 0.411 | 0.255 | 0.119 | 0.000 |
| 130 | 2.291 | 1.553 | 1.123 | 0.827 | 0.598 | 0.412 | 0.255 | 0.119 | 0.000 |
| 140 | 2.291 | 1.553 | 1.123 | 0.827 | 0.598 | 0.412 | 0.255 | 0.119 | 0.000 |
| 160 | 2.292 | 1.553 | 1.124 | 0.827 | 0.598 | 0.412 | 0.255 | 0.119 | 0.000 |

The results presented in Fig. 1 are the dimensionless pressure at the wellbore at different dimensionless time for the case of constant pressure outer boundary condition. When a reservoir is opened for production, a pressure disturbance is created in the reservoir from the wellbore. This disturbance is not only felt at the wellbore but it travels through the entire reservoir formation to the external boundary. Therefore, Tables 2 and 3 shows the dimensionless pressure at different points within and outside the wellbore of the reservoir against their corresponding dimensionless time.

8. CONCLUSION

This paper has been able to present the pressure distribution across a bounded circular reservoir assumed to have constant terminal rate at the wellbore. The diffusivity equation was used to analyse the pressure in the system. It was shown from Figs. 1 that the dimensionless pressure increases drastically immediately this flow regime is attained. But as time increases, the dimensionless pressure variation flattens out asymptotically. The results obtained from this analysis showed that there was a strong correlation with the results obtained from the Van Everdigen and Hurst. It is important to note that the Van Everdigen and Hurst solutions only state the pressure at the wellbore at a particular time but this work predicts the pressure variation in the entire reservoir from the wellbore to the external boundary at the same time. These where shown in Tables 2 and 3 and it was noticed that the pressure decreases from the wellbore to the external boundary of the reservoir. Therefore the Finite element method has been used to approximate not only the values of the wellbore pressures for bounded circular reservoirs but also pressure outside the wellbore to the external reservoir boundaries.

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